

# ME-662 CONVECTIVE HEAT AND MASS TRANSFER

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LECTURE-8 SIMILARITY SOLN TO TEMP BL - I

# LECTURE-8 SIM SOLN TO TEMP BL - I

- 1 Condition for Existence of Similarity Solutions
- 2 Similarity Equation and Boundary Conditions

# BL Energy Equation L8( $\frac{1}{11}$ )

$$\rho C_p \left[ u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right] = k \frac{\partial^2 T}{\partial y^2} + \mu \left( \frac{\partial u}{\partial y} \right)^2 + u \frac{dp_\infty}{dx} + \dot{Q}_{chem} + \dot{Q}_{rad} \quad (1)$$

## Source Terms and Boundary Conditions:

- 1  $\dot{Q}_{chem}$  and  $\dot{Q}_{rad}$  are presently neglected
- 2  $u dp_\infty/dx$  is important only in high-speed gas flows - presently neglected
- 3 at  $y = 0$ ,  $T = T_w(x)$  ( Wall Temperature )
- 4 as  $y \rightarrow \infty$ ,  $T = T_\infty$  ( Constant Free Stream Temperature )

# Development of Similarity Eqn - L8( $\frac{2}{11}$ )

Define

$$T_w(x) - T_\infty = G(x) \quad (2)$$

$$\theta(\eta) = \frac{T(x, y) - T_\infty}{T_w(x) - T_\infty} \quad \eta = y \sqrt{\frac{U_\infty}{\nu x}} \quad (3)$$

Then, the energy eqn will read as

$$\left[ u \frac{\partial \theta}{\partial x} + v \frac{\partial \theta}{\partial y} \right] + \frac{u\theta}{G} \frac{dG}{dx} = \alpha \frac{\partial^2 \theta}{\partial y^2} + \frac{\nu}{C_p (T_w - T_\infty)} \left( \frac{\partial u}{\partial y} \right)^2 \quad (4)$$

Each term is now represented in **similarity variables**

# Similarity Variables - L8( $\frac{3}{11}$ )

Recall the following definitions

$$U_{\infty} = C x^m$$

$$\eta = y \sqrt{\frac{U_{\infty}}{\nu x}} \quad \psi = f(\eta) n(x) \quad n(x) = \sqrt{\nu U_{\infty} x}$$

$$u = U_{\infty} f' \quad v = -\frac{\partial \psi}{\partial x} = -\left[ f' n(x) \frac{\partial \eta}{\partial x} + f \frac{dn}{dx} \right]$$

$$\frac{\partial \theta}{\partial x} = \theta' \frac{\partial \eta}{\partial x} \quad \frac{\partial \theta}{\partial y} = \theta' \frac{\partial \eta}{\partial y} = \theta' \sqrt{\frac{U_{\infty}}{\nu x}}$$

$$\frac{\partial^2 \theta}{\partial y^2} = \theta'' \frac{U_{\infty}}{\nu x} \quad \left( \frac{\partial u}{\partial y} \right)^2 = \frac{U_{\infty}^3}{\nu x} (f'')^2$$

Substitution gives ( see next slide )

# Similarity Equation - I - L8( $\frac{4}{11}$ )

$$\begin{aligned} f' \theta' \frac{\partial \eta}{\partial x} - \frac{\theta'}{n} \left[ f' n \frac{\partial \eta}{\partial x} + f \frac{dn}{dx} \right] + \frac{f' \theta}{G} \frac{dG}{dx} \\ = \frac{\theta''}{Pr x} + \frac{U_\infty^2}{Cp(T_w - T_\infty)} \frac{(f'')^2}{x} \end{aligned}$$

or, upon simplification and multiplication by x

$$f' \theta \left( \frac{x}{G} \frac{dG}{dx} \right) - f \theta' \left( \frac{x}{n} \frac{dn}{dx} \right) = \frac{\theta''}{Pr} + 2 Ec_x (f'')^2$$

where **Eckert Number**  $Ec_x = (U_\infty^2/2)/(Cp(T_w - T_\infty))$ .

It can be shown that  $(x/n)(dn/dx) = (m+1)/2$ .

# Similarity Equation - II - L8( $\frac{5}{11}$ )

Hence the similarity equation will read as

$$\theta'' + Pr \left[ \left( \frac{m+1}{2} \right) f \theta' - f' \theta \left( \frac{x}{G} \frac{dG}{dx} \right) + 2 Ec_x (f'')^2 \right] = 0 \quad (5)$$

Similarity solutions are possible **only when**

- 1  $(x/G)(dG/dx) = \text{constant}$  ( $\gamma$ , say) or  
 $G(x) = T_w(x) - T_\infty = \Delta T_{ref} x^\gamma$
- 2  $Ec_x = (U_\infty^2(x)/2)/(Cp(T_w(x) - T_\infty)) = \text{constant}$  or

$$Ec_x = \left( \frac{C^2}{2Cp \Delta T_{ref}} \right) \left( \frac{x^{2m}}{x^\gamma} \right) = \text{constant}$$

- 3 Hence,  $\gamma = 2m$  when  $Ec_x \neq 0$  ( or, when viscous dissipation is accounted )

# Final Similarity Equation - L8( $\frac{6}{11}$ )

Hence the final similarity equation will read as

$$\theta'' + Pr \left[ \left( \frac{m+1}{2} \right) f \theta' - \gamma f' \theta + 2 Ec (f'')^2 \right] = 0 \quad (6)$$

where  $Ec = (U_\infty^2(x)/2)/(Cp \Delta T_{ref} x^\gamma)$ . If  $Ec \neq 0$ ,  $\gamma = 2m$

The Boundary Conditions are:

$$\theta(0) = 1 \quad \text{and} \quad \theta(\infty) = 0$$

Solution:  $\theta(\eta) = F(m, B_f, Pr, \gamma, Ec)$  If  $Ec \neq 0$ ,  $\gamma = 2m$



# Shooting Method - L8( $\frac{7}{11}$ )

The 2nd order equation is split into two 1st order ODEs

$$\frac{d\theta}{d\eta} = \theta' \quad \text{with} \quad \theta(0) = 1 \quad (\text{known}) \quad (7)$$

$$\frac{d\theta'}{d\eta} = \theta'' = -Pr \left[ \left(\frac{m+1}{2}\right) f \theta' - \gamma f' \theta + 2Ec (f'')^2 \right]$$

with  $\theta'(0)$  (unknown) (8)

- 1 Solution of Velocity Boundary Layer gives  $f, f', f''$
- 2 Then,  $\theta'(0)$  is guessed and the two equations are solved by R-K method from  $\eta = 0$  to  $\eta = \eta_{max}$ .
- 3 At each iteration, BC  $\theta(\eta_{max}) \rightarrow 0$  is checked.
- 4 If NOT satisfied,  $\theta'(0)$  is revised

# Output Parameters - I - L8( $\frac{8}{11}$ )

- 1 The *Physical Thickness*  $\Delta$  is notionally associated with value of  $y$  where  $\theta(\eta_{max}) \simeq 0.01$ .
- 2 *Enthalpy Thickness*  $\Delta_2$  is defined as

$$\Delta_2 = \int_0^{\infty} \frac{\rho C_p u (T - T_{\infty})}{\rho_{\infty} C_{p_{\infty}} U_{\infty} (T_w - T_{\infty})} dy \quad (9)$$

- 3 Dimensionless Form ( Uniform Property )

$$\Delta_2^* = \frac{\Delta_2}{x} Re_x^{0.5} = \int_0^{\eta_{max}} f' \theta d\eta \quad (10)$$

# Output Parameters - II - L8( $\frac{9}{11}$ )

Local H T Coef, Nusselt and Stanton Numbers

$$h_x = \frac{q_w}{T_w - T_\infty} = -\frac{k (\partial T / \partial y)_{y=0}}{T_w - T_\infty} = -k \sqrt{\frac{U_\infty}{\nu x}} \theta' (0) \quad (11)$$

$$Nu_x = \frac{h_x x}{k} = -Re_x^{0.5} \theta' (0) \quad (12)$$

$$St_x = \frac{h_x}{\rho C_p U_\infty} = \frac{Nu_x}{Re_x Pr} \quad (13)$$

$Nu_x, St_x = F(m, B_f, Pr, \gamma, Ec)$  If  $Ec \neq 0$ ,  $\gamma = 2m$

$$\overline{Nu} = \frac{\bar{h} x}{k} = \left(\frac{2}{m+1}\right) Nu_L \quad \bar{h} = \frac{1}{L} \int_0^L h_x dx \quad (14)$$

# Typical Profiles - L8( $\frac{10}{11}$ )

Flat Plate, No Suction/Blowing, Const Wall Temp, No Viscous Dissipation (  $Pr = 1$  )

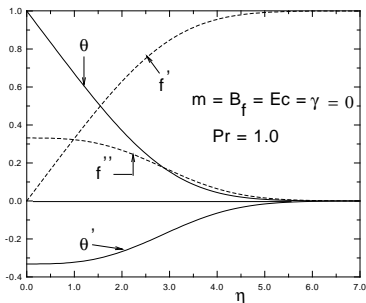


Figure:  $f''' + 0.5 f f'' = 0$  and  $\theta'' + 0.5 f \theta' = 0$  .

$f(0) = f'(0) = 0, \theta(0) = 1$  and  $f'(\infty) = 1, \theta(\infty) = 0$  .

Hence,  $\theta(\eta) = 1 - f'(\eta)$  - Perfect Analogy Between Heat and Momentum Transfer  $\Delta^* = \delta^*, \Delta_2^* = \delta_2^*$  and  $-\theta'(0) = f''(0)$

# Moderate Pr Numbers - L8( $\frac{11}{11}$ )

(  $m = B_f = \gamma = Ec = 0$  )

Pr	0.7	1.0	5.0	10.0	25.0
$-\theta'(0)$	0.291	0.330	0.572	0.721	0.976
$\Delta^*$	5.60	4.92	2.73	2.15	1.59
$\Delta_2^*$	0.834	0.663	0.231	0.146	0.0796

$-\theta'(0) = Nu_x / Re_x^{0.5}$  can be correlated as

$$Nu_x = 0.332 Re_x^{0.5} Pr^{0.33}$$

Very good agreement with Experimental data.

$\Delta^*$  decreases with increase in Pr.  $\Delta^* = \delta^*$  for Pr = 1 .

In the next lecture, effects of other parameters are considered